

Tribology of a CVT traction drive

Supervisor: prof.dr.ir. M. Steinbuch

Coaches: dr.ir. Roëll van Druten

dr.ir. Bas Vroemen

ir. Irmak Aladagli

dr. ir. Theo Hofman

Jeroen van den Brand

Lichao Pan



TU / **e** Technische Universiteit
Eindhoven
University of Technology

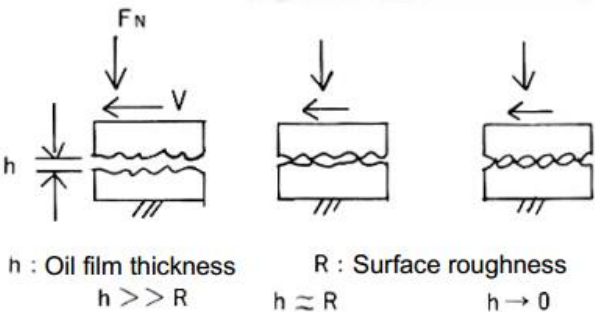
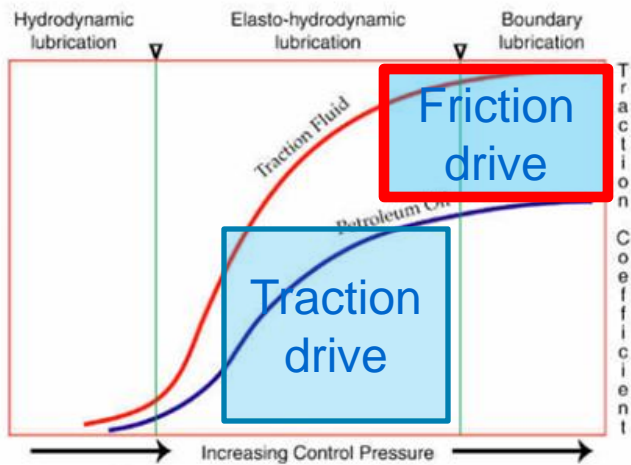
Where innovation starts

Structure

- **Background**
- **Introduction**
- **Traction drive modeling**
- **NuVinci analysis with model**

Aim: predict efficiency of NuVinci transmission, which works as a traction drive

Background: CVT types

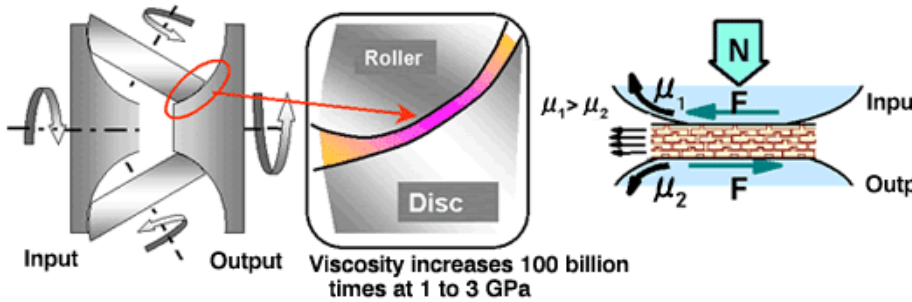


Manuel E. Joaquim. Ehls: The secret behind cvts. Technical report, Findett Corporation.

Types	Product	
Pulley based CVTs	<p>LuK chain CVT</p>	<p>Bosch pushbelt CVT</p>
Toroidal CVT	<p>Nissan Extroid toroidal CVT</p>	
Ball based CVTs	<p>Kopp Variator</p>	<p>NuVinci CVP</p>

Introduction: traction drive fluid

TDF for Lubricating Power



Traction Force $F = \text{Traction coefficient } \mu \times \text{Weight } N$
 The higher the traction coefficient, the higher the transmission capacity

TDF as a Functional Part of the Transmission

LTD Nissan Motor Co. Extroid cvt: for application to rear-wheel-drive cars powered by large engines.

Traction fluid behaves in a **solid** like manner under high contact stress
 (order of 1 [GPa], 10,000 times atmospheric pressure)

The mountains flowed before the Lord



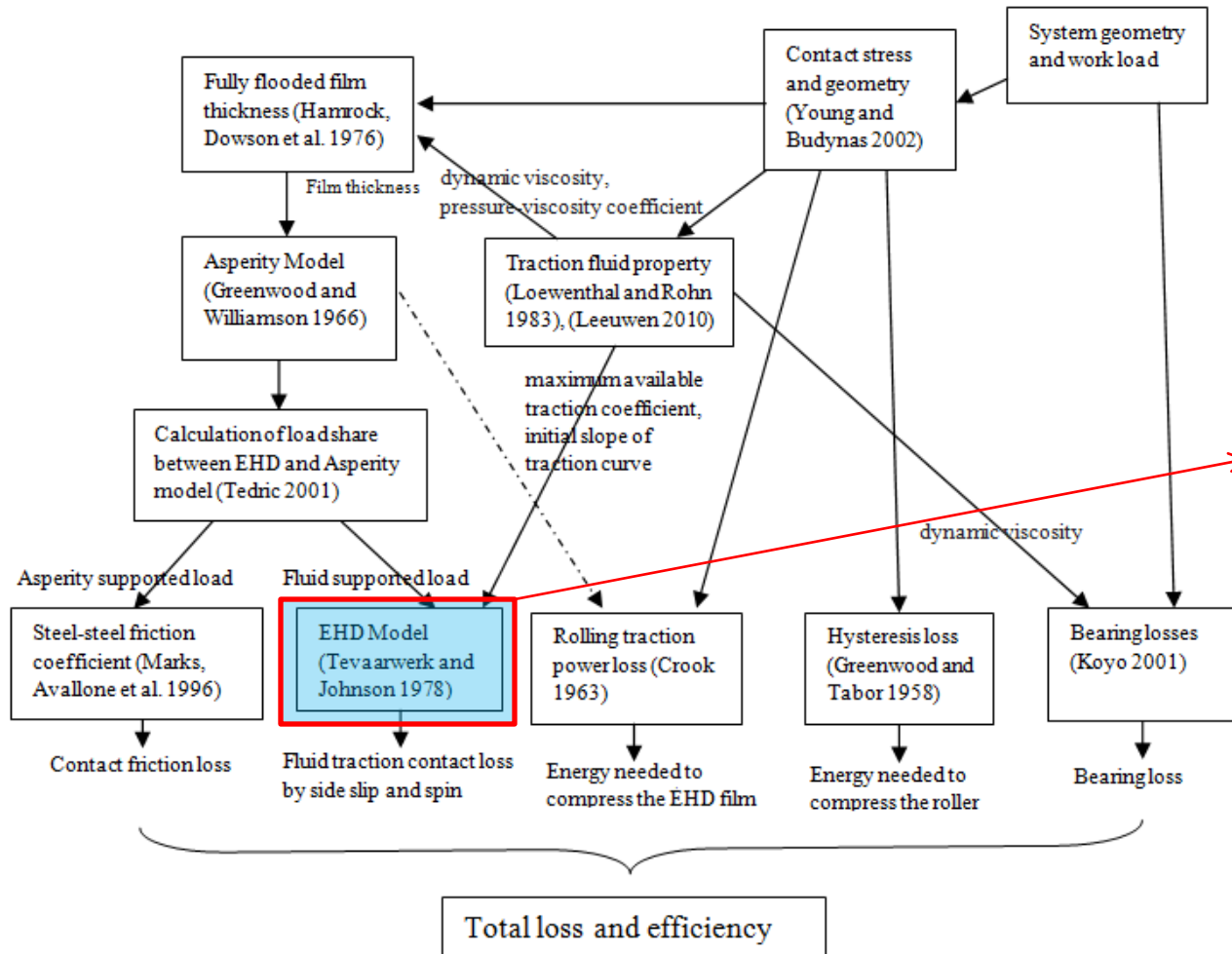
relaxation time

$$De = \frac{t_c}{t_p}$$

Observation time

Deborah (1200BC – 1124 BC):
 a prophetess of the God of the Israelites

Modeling



Johnson-Tevaarwerk model gives good predictions at **large Deborah number**:

- Large contact pressure
- Short transit time

Modeling: Johnson-Tevaarwerk model

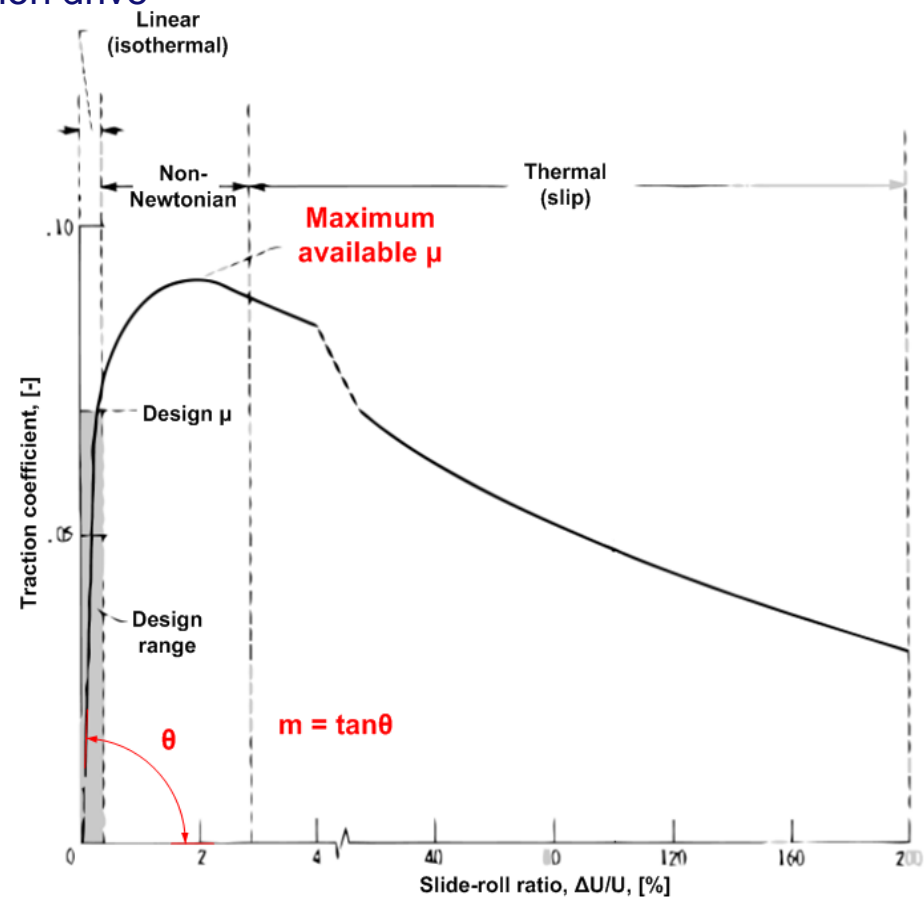
- Developed in 1978
- Based on limited shear stress theory
- Describing the elastic and plastic behavior in traction drive
- Two key parameters:

$$G = \frac{3 m N h}{8 a^2 b}$$

elastic shear modulus

$$\tau_c = \frac{\mu N}{\pi a b}$$

average limiting shear strength



Modeling: Johnson-Tevaarwerk model

Dimensionless parameters

- Dimensionless slip in rolling direction:
- Dimensionless slip in transverse direction:
- Dimensionless spin:
- Dimensionless traction force in rolling direction:
- Dimensionless traction force in transverse direction:
- Dimensionless torque perpendicular to contact area:

$$J_1 = C \frac{\Delta U}{U}$$

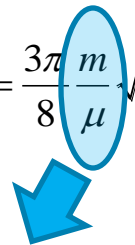
$$J_2 = C \frac{\Delta V}{U}$$

$$J_3 = C \frac{\omega_s \sqrt{ab}}{U}$$

$$J_4 = \frac{F_x}{\tau_c \pi ab} = \frac{F_x}{\mu N}$$

$$J_5 = \frac{F_y}{\tau_c \pi ab} = \frac{F_y}{\mu N}$$

$$J_6 = \frac{T_s}{\mu N \sqrt{ab}}$$

$$C = \frac{G \sqrt{ab}}{\tau_c h_c} = \frac{3\pi}{8} \frac{m}{\mu} \sqrt{k}$$


Loewenthal and Rohn investigated 334 traction drives experimental results, a regression model was given based on Santotrac 50.

Losses

Slip loss: $P_{slip} = \Delta U F_x = J_1 J_4 \frac{U \mu N}{C}$

Spin loss: $P_{spin} = \omega_s T = J_3 J_6 \frac{U \mu N}{C}$

Transferred power: $P_{in} = U F_x = \frac{J_4 U \mu N}{sf}$

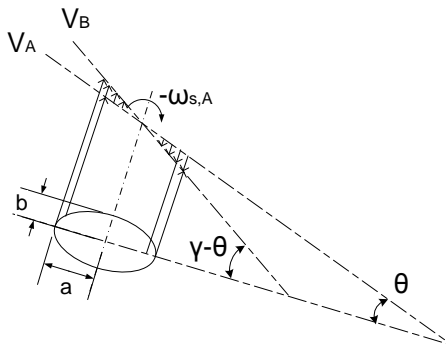
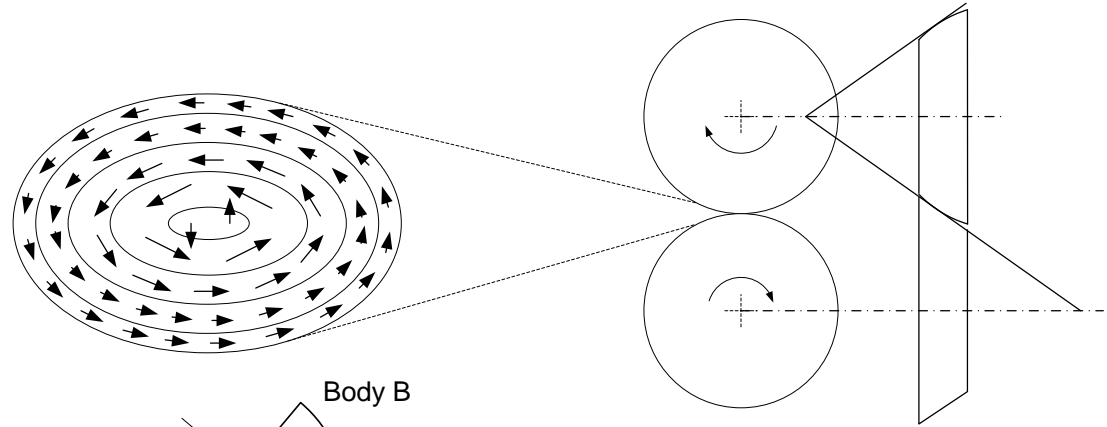


$$\frac{P_{slip} + P_{spin}}{P_{in}} = \frac{J_1 J_4 + J_3 J_6}{J_4 C} sf$$

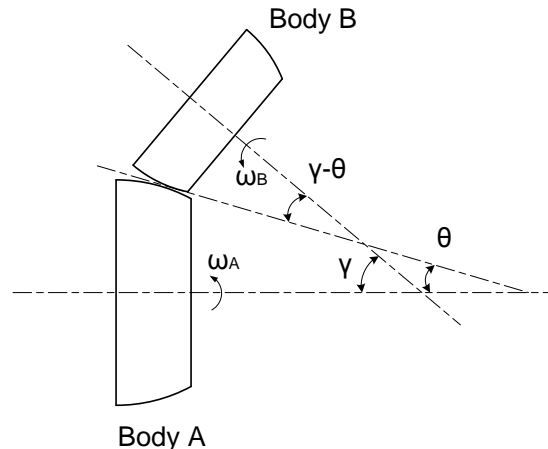
Model parameter: Spin, critical factor

Spin:

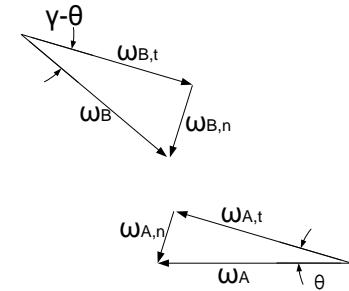
“a difference in the angular velocity vector between the bodies in the direction normal to the contact”



Spin contact pattern on roller A



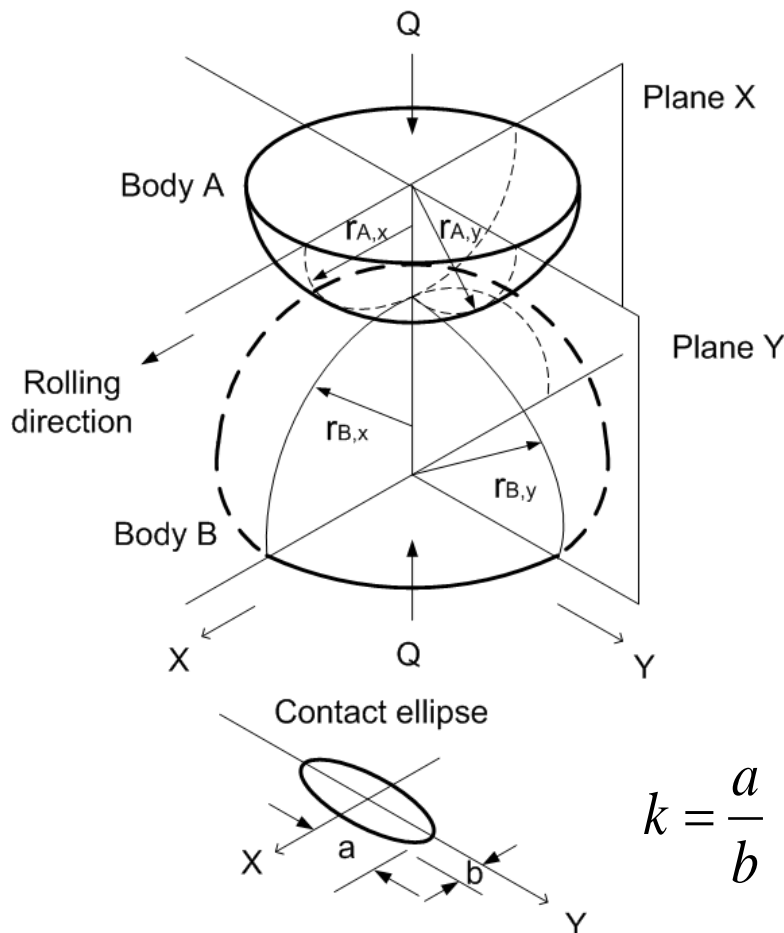
Contact geometry for spin calculations



Angular velocity diagram

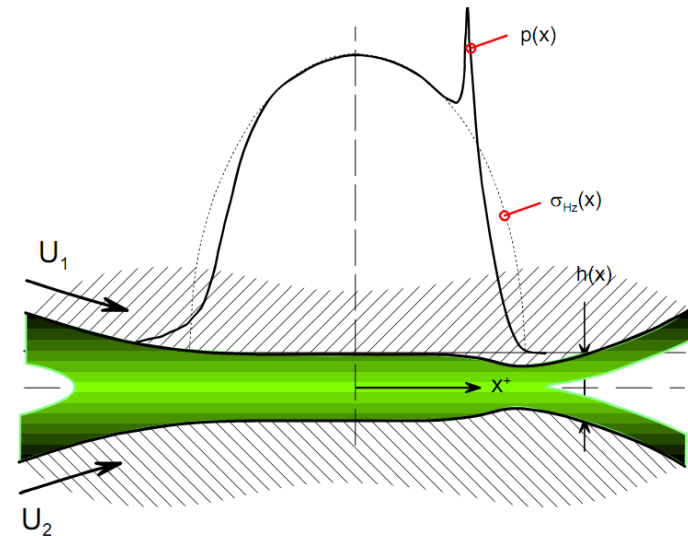
Model parameter: contact size and stress

Hertzian contact



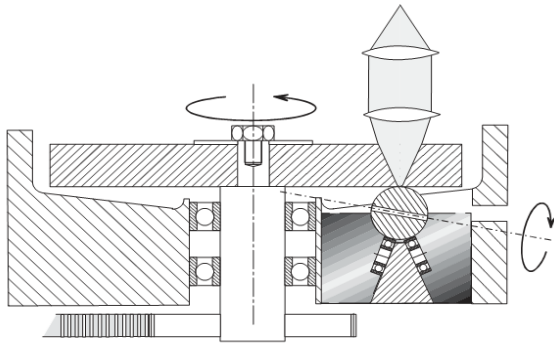
$$k = \frac{a}{b}$$

Film thickness

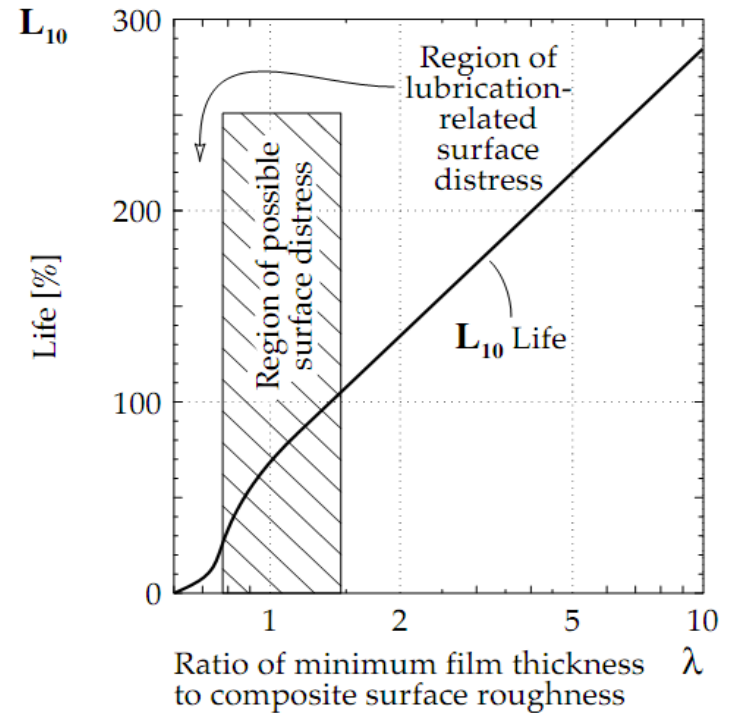
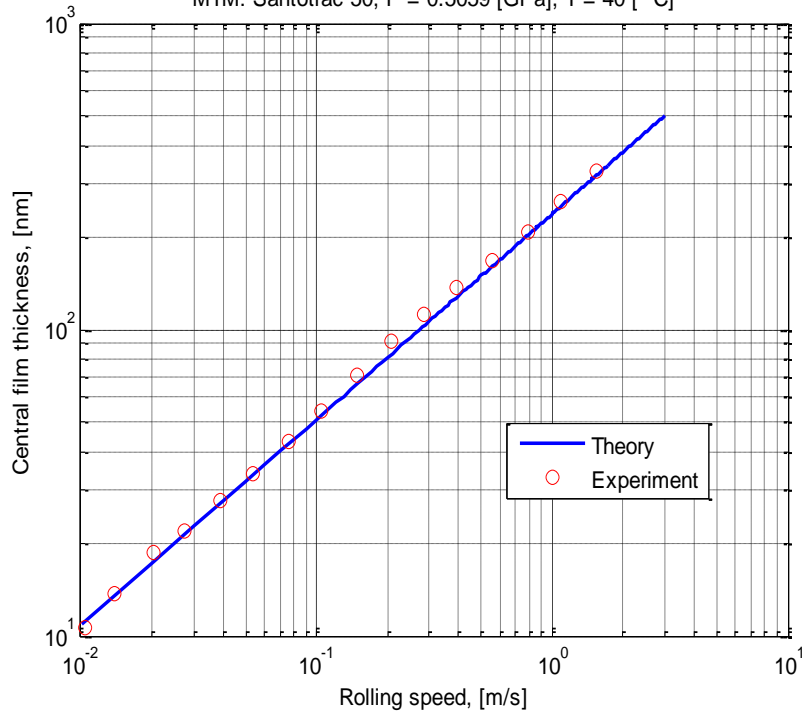


$$H_c = 2.69U^{0.67}G^{0.53}W^{-0.067}(1 - 0.61e^{-0.73k})$$

Modeling: film thickness and surface roughness



MTM: Santotrac 50, P = 0.5059 [GPa], T = 40 [° C]

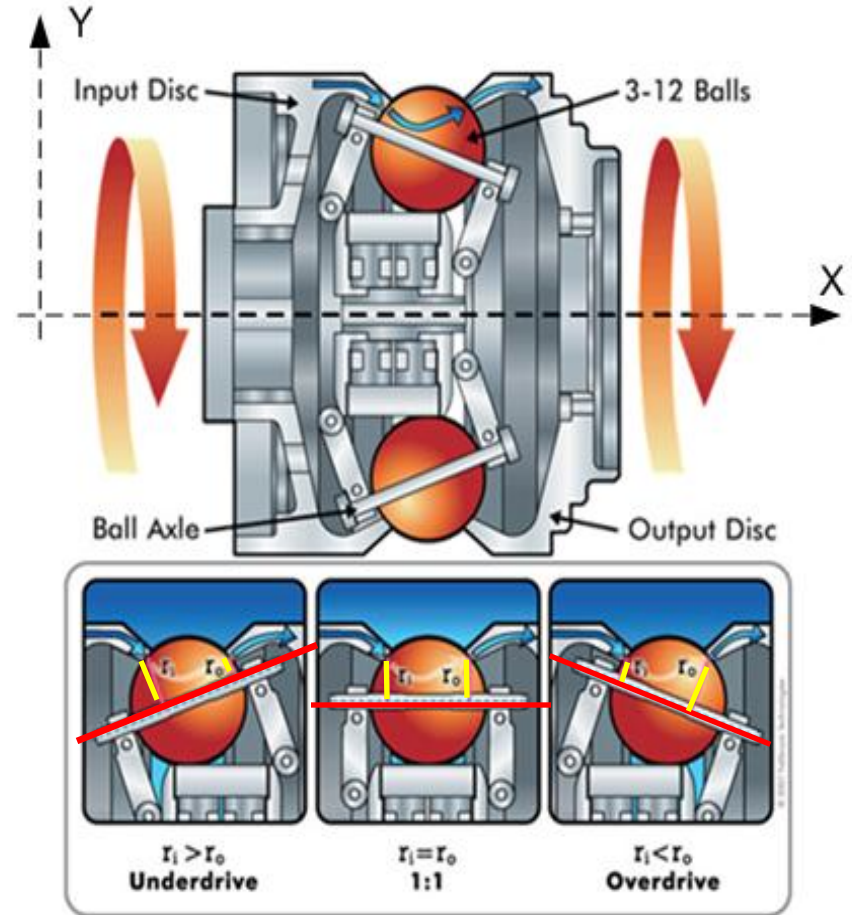


$$\lambda = \frac{h_c}{\sqrt{\sigma_A^2 + \sigma_B^2}}$$

NuVinci analysis: Introduction

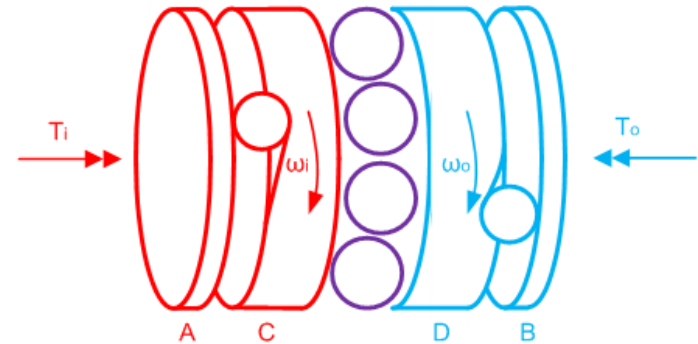
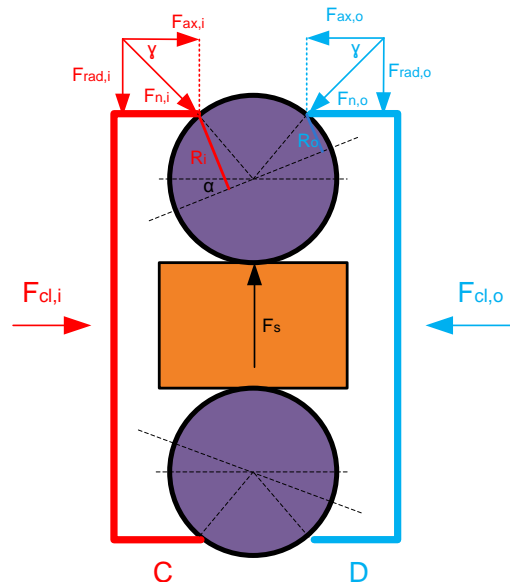
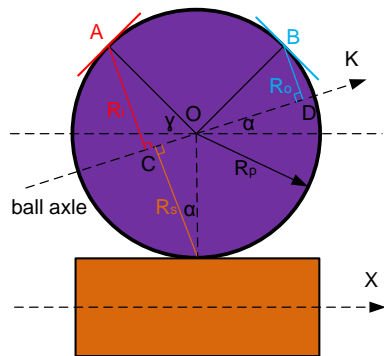
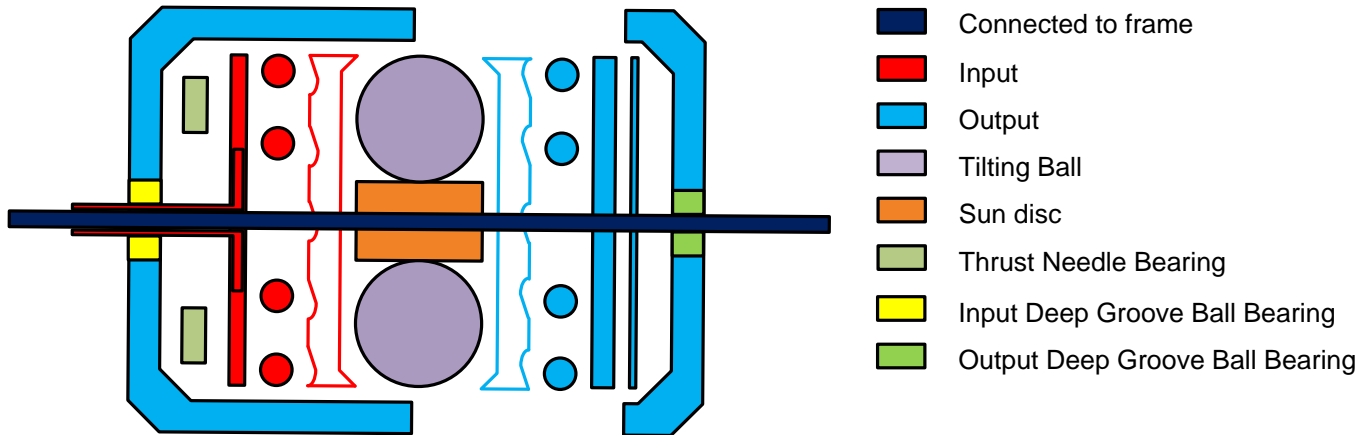
The NuVinci hub is a new concept introduced in 2007:

- Simple
- Compact
- Continuous shifting
- Non-hydraulic

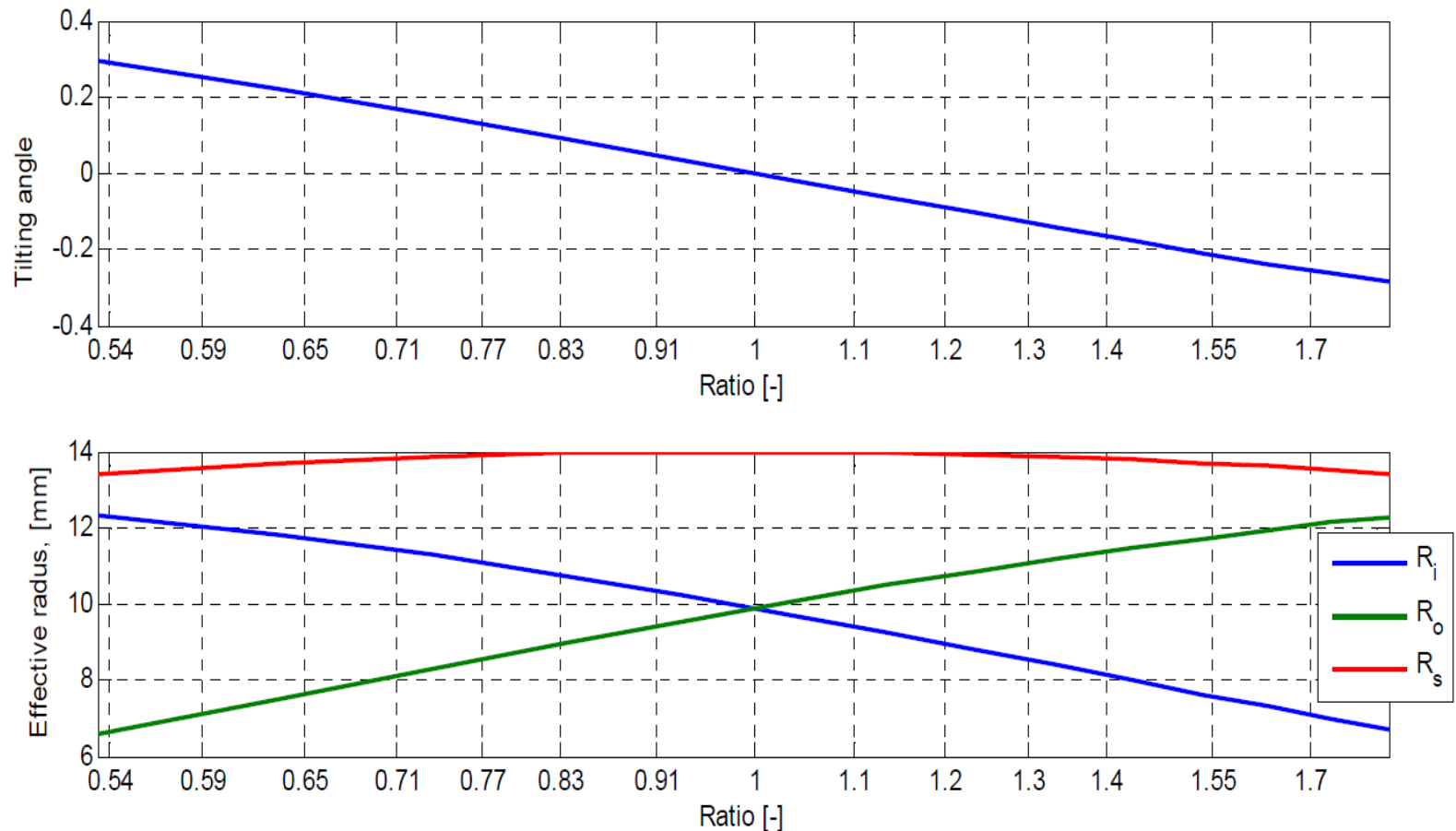


Batavus' bike catalog 2012: Vivente NuVinci

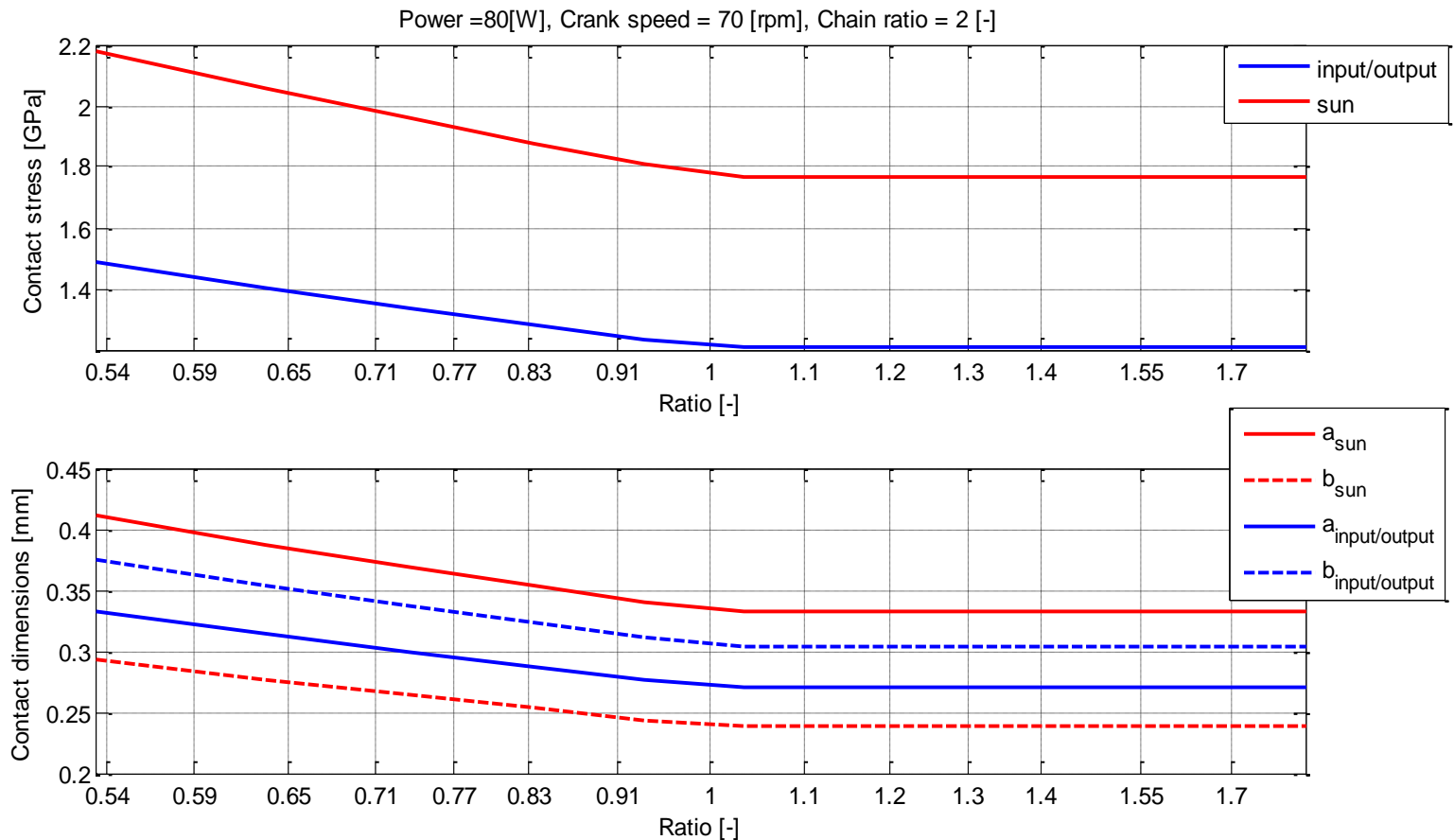
NuVinci analysis: geometry and force



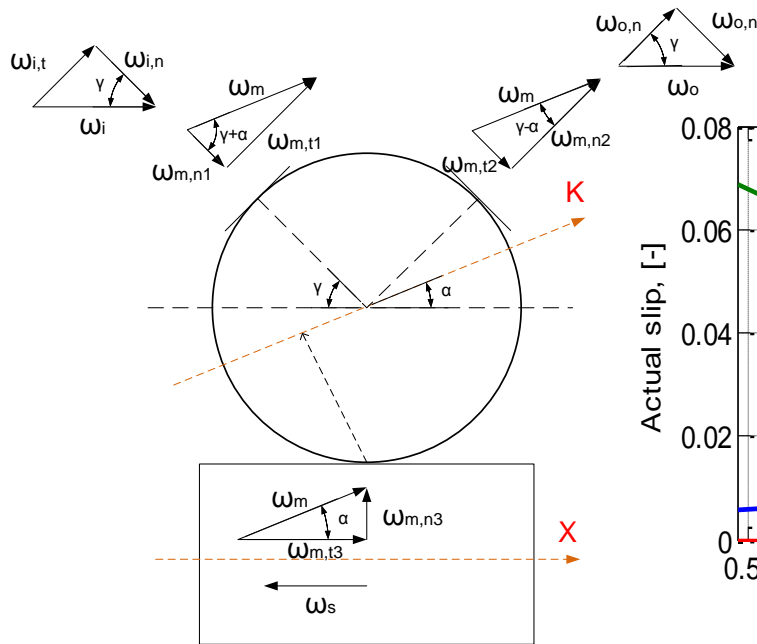
NuVinci analysis: ratio and tilting angle



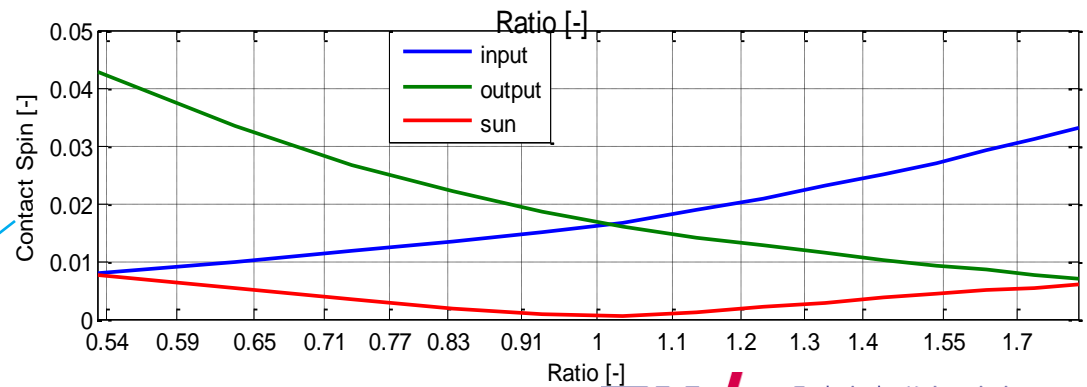
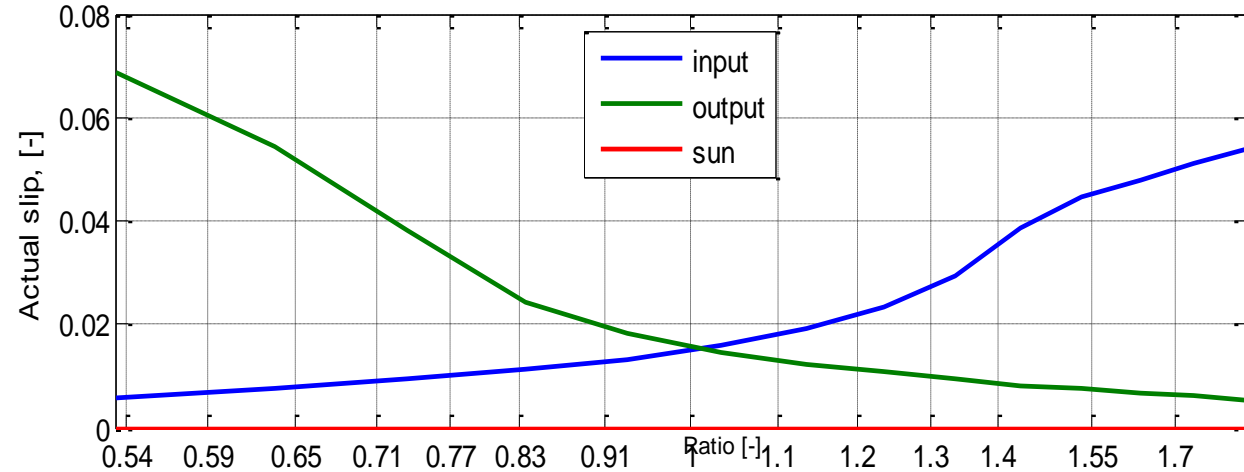
NuVinci analysis: contact stress and size



NuVinci analysis: Spin and slip

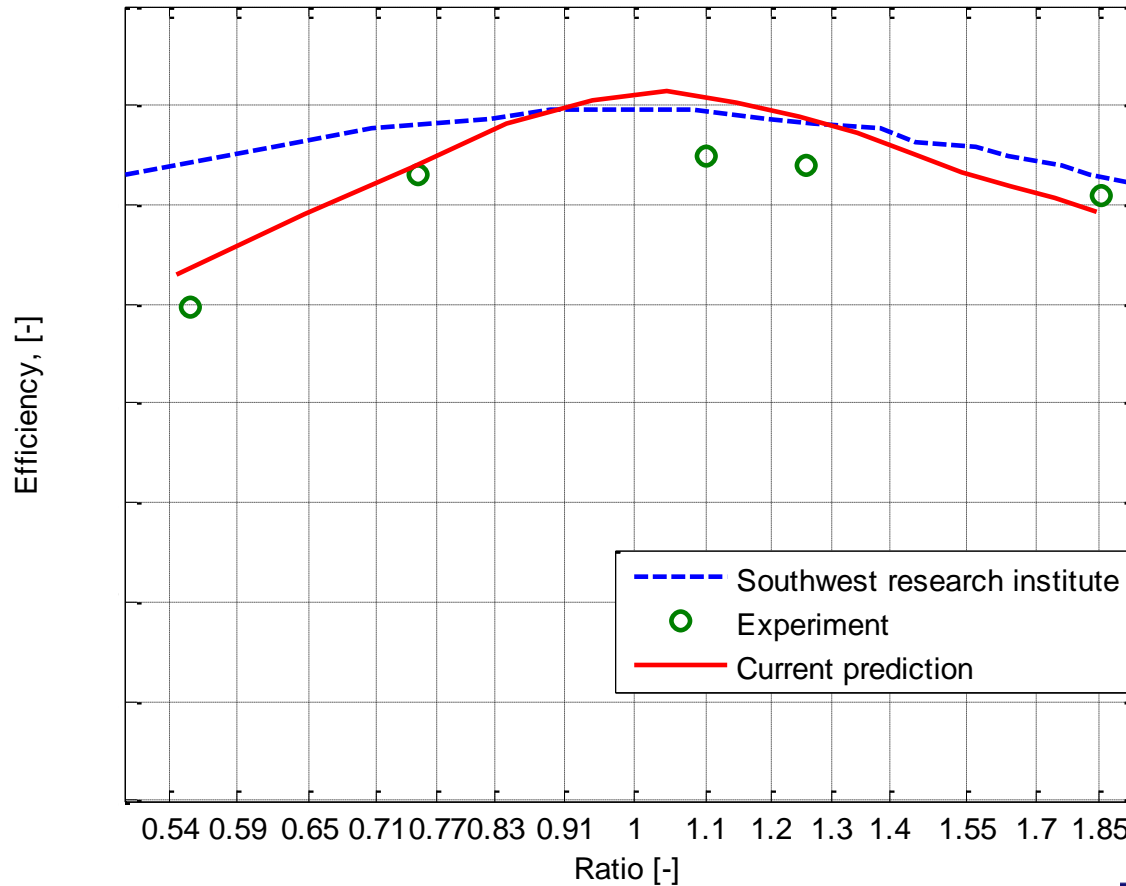


Power = 80[W], Crank speed = 70 [rpm], Chain ratio = 2 [-]

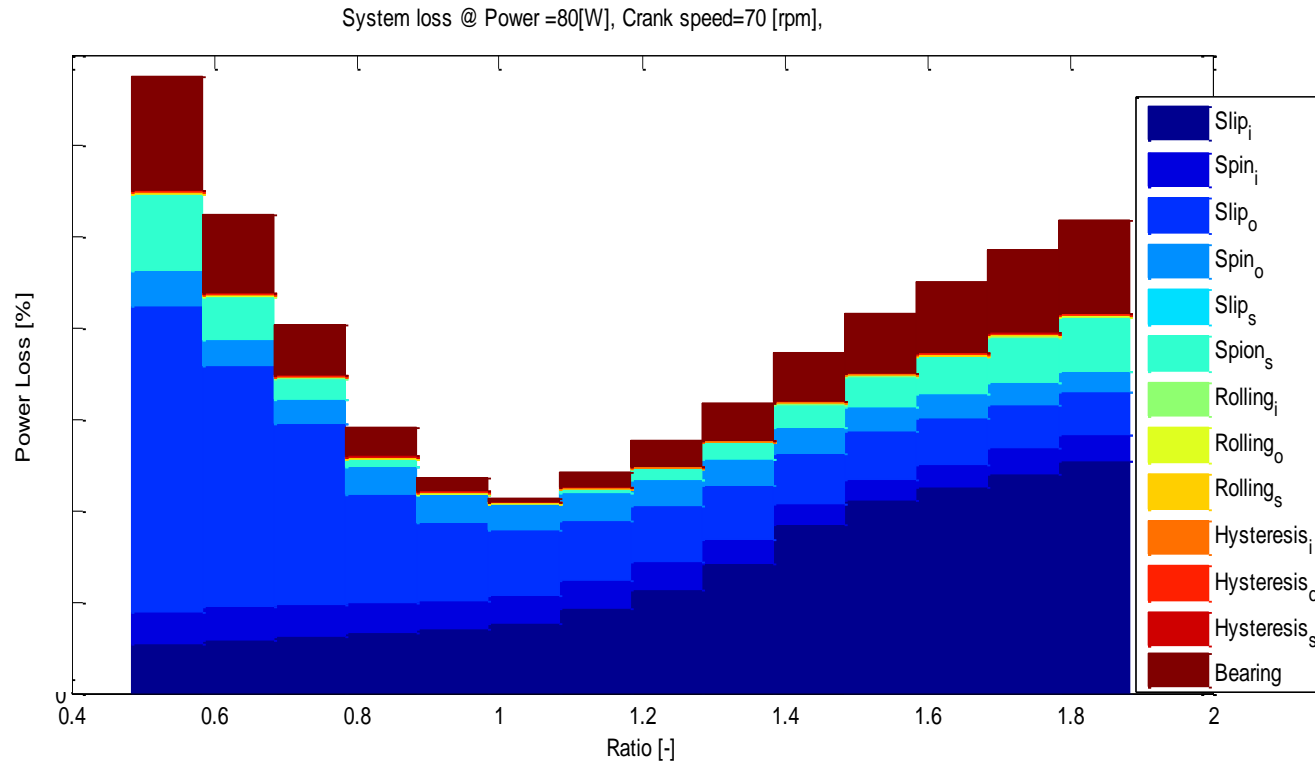


$$\frac{\omega_s \sqrt{ab}}{U}, \text{ non-dimensional}$$

NuVinci analysis: system efficiency



NuVinci analysis: system efficiency



NuVinci analysis: Conclusion

- **Slip losses are the main source of system losses;**
- **However, they are very sensitive to contact spin level;**
- **Thus eliminating the spin on contact interface is the key point to improve a traction drive efficiency.**

Questions?

